



COURSE OVERVIEW PE0127 Operations Abnormalities & Plant Upset

Course Title

Operations Abnormalities & Plant Upset

Course Date/Venue

September 07-11, 2025/TBA Meeting Room,
Hilton Kuwait Resort, Mangaf, Kuwait City,
Kuwait

Course Reference

PE0127

Course Duration/Credits

Five days/3.0 CEUs/30 PDHs

Course Description



This practical and highly-interactive course includes various practical sessions and exercises. Theory learnt will be applied using our state-of-the-art simulators.

Managing Manpower effectively and assess risk properly during plant upset are key effective factors when reacting with incidents. Incidents may start minor and become major by wrong reaction and wrong decisions. The aim of this course is to make everybody involved in the operations know exactly what to do. The incident itself may cause a certain loss, but with wrong reaction it became a massive loss. Understanding operation, effective emergency/contingency plan, rules of each one within emergency plan and makes emergency tools ready and in operational condition are the main aims of this course. One approach to overcome any incident development is to prepare yourself and emergency team to treat incidents situation professionally.

Upon review of several incidents, two common causes were identified that contributed to those incidents. The causes are improper management of manpower during upset conditions and improper risk assessment of activities to be executed or stop doing. However, on close examination the trained emergency team and correct managing of the incident besides using correct emergency tools will minimize the loss and accidents consequences.



Effective training is the necessary foundation for the successful implementation of optimum emergency managing condition and optimum consequences minimizing. This course will train participants on managing risk & manpower during plant upset to save lives, assets and company reputations.

Course Objectives

Upon the successful completion of this course, each participant will be able to:-

- Manage manpower effectively and assess risk properly during the abnormalities of the operations and plant upset
- Assess staffing level in abnormal situations and distribute manpower during plant upset conditions
- Manage shift teams, assess risk of non-routine activities and manage operational crisis
- Identify risks in the process and describe the roles, responsibilities and procedures in emergency management
- Use the risk assessment process and have enough skills in monitoring and auditing the emergency tools
- Recognize the training requirements for process emergency handling including emergency team building
- Discuss the various skills that will be acquired in controlling emergency management using different scenarios and matrix
- Identify the common mistakes during emergencies and employ the preventive measures

Exclusive Smart Training Kit - H-STK®



Participants of this course will receive the exclusive “Haward Smart Training Kit” (H-STK®). The H-STK® consists of a comprehensive set of technical content which includes **electronic version** of the course materials conveniently saved in a **Tablet PC**.

Who Should Attend

This course provides an overview of all significant aspects and considerations of operations abnormalities and plant upset for superintendents, supervisors and foremen in various departments of process plants (production, operations, maintenance, utility, etc.). Further, the course is suitable for emergency teams, managers, supervisors and other technical staff.

Course Fee


US\$ 5,500 per Delegate. This rate includes H-STK® (Haward Smart Training Kit), buffet lunch, coffee/tea on arrival, morning & afternoon of each day.

Course Certificate(s)

Internationally recognized certificates will be issued to all participants of the course who completed a minimum of 80% of the total tuition hours.

Certificate Accreditations


Certificates are accredited by the following international accreditation organizations: -

- 
The International Accreditors for Continuing Education and Training (IACET - USA)

Haward Technology is an Authorized Training Provider by the International Accreditors for Continuing Education and Training (IACET), 2201 Cooperative Way, Suite 600, Herndon, VA 20171, USA. In obtaining this authority, Haward Technology has demonstrated that it complies with the **ANSI/IACET 2018-1 Standard** which is widely recognized as the standard of good practice internationally. As a result of our Authorized Provider membership status, Haward Technology is authorized to offer IACET CEUs for its programs that qualify under the **ANSI/IACET 2018-1 Standard**.

Haward Technology's courses meet the professional certification and continuing education requirements for participants seeking **Continuing Education Units (CEUs)** in accordance with the rules & regulations of the International Accreditors for Continuing Education & Training (IACET). IACET is an international authority that evaluates programs according to strict, research-based criteria and guidelines. The CEU is an internationally accepted uniform unit of measurement in qualified courses of continuing education.

Haward Technology Middle East will award **3.0 CEUs** (Continuing Education Units) or **30 PDHs** (Professional Development Hours) for participants who completed the total tuition hours of this program. One CEU is equivalent to ten Professional Development Hours (PDHs) or ten contact hours of the participation in and completion of Haward Technology programs. A permanent record of a participant's involvement and awarding of CEU will be maintained by Haward Technology. Haward Technology will provide a copy of the participant's CEU and PDH Transcript of Records upon request.

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British Accreditation Council (BAC)

Haward Technology is accredited by the **British Accreditation Council for Independent Further and Higher Education** as an **International Centre**. BAC is the British accrediting body responsible for setting standards within independent further and higher education sector in the UK and overseas. As a BAC-accredited international centre, Haward Technology meets all of the international higher education criteria and standards set by BAC.

Accommodation

Accommodation is not included in the course fees. However, any accommodation required can be arranged at the time of booking.



Course Instructor(s)

This course will be conducted by the following instructor(s). However, we have the right to change the course instructor(s) prior to the course date and inform participants accordingly:



Mr. Mervyn Frampton is a **Senior Process Engineer** with over **30 years** of industrial experience within the **Oil & Gas, Refinery, Petrochemical** and **Utilities** industries. His expertise lies extensively in the areas of **Process Troubleshooting, Distillation Towers, Fundamentals of Distillation** for Engineers, **Distillation Operation and Troubleshooting, Advanced Distillation Troubleshooting, Distillation Technology, Vacuum Distillation, Distillation Column Operation & Control, Oil Movement Storage & Troubleshooting, Process Equipment Design, Applied Process Engineering Elements, Process Plant Optimization, Revamping & Debottlenecking, Process Plant Troubleshooting & Engineering Problem Solving, Process Plant Monitoring, Catalyst Selection & Production Optimization, Operations Abnormalities & Plant Upset, Process Plant Start-up & Commissioning, Clean Fuel Technology & Standards, Flare, Blowdown & Pressure Relief Systems, Oil & Gas Field Commissioning Techniques, Pressure Vessel Operation, Gas Processing, Chemical Engineering, Process Reactors Start-Up & Shutdown, Gasoline Blending for Refineries, Urea Manufacturing Process Technology, Continuous Catalytic Reformer (CCR), De-Sulfurization Technology, Advanced Operational & Troubleshooting Skills, Principles of Operations Planning, Rotating Equipment Maintenance & Troubleshooting, Hazardous Waste Management & Pollution Prevention, Heat Exchangers & Fired Heaters Operation & Troubleshooting, Energy Conservation Skills, Catalyst Technology, Refinery & Process Industry, Chemical Analysis, Process Plant, Commissioning & Start-Up, Alkylation, Hydrogenation, Dehydrogenation, Isomerization, Hydrocracking & De-Alkylation, Fluidized Catalytic Cracking, Catalytic Hydrodesulphuriser, Kerosene Hydrotreater, Thermal Cracker, Catalytic Reforming, Polymerization, Polyethylene, Polypropylene, Pilot Water Treatment Plant, Gas Cooling, Cooling Water Systems, Effluent Systems, Material Handling Systems, Gasifier, Gasification, Coal Feeder System, Sulphur Extraction Plant, Crude Distillation Unit, Acid Plant Revamp and Crude Pumping. Further, he is also well-versed in HSE Leadership, Project and Programme Management, Project Coordination, Project Cost & Schedule Monitoring, Control & Analysis, Team Building, Relationship Management, Quality Management, Performance Reporting, Project Change Control, Commercial Awareness and Risk Management.**

During his career life, Mr. Frampton held significant positions as the **Site Engineering Manager, Senior Project Manager, Process Engineering Manager, Project Engineering Manager, Construction Manager, Site Manager, Area Manager, Procurement Manager, Factory Manager, Technical Services Manager, Senior Project Engineer, Process Engineer, Project Engineer, Assistant Project Manager, Handover Coordinator** and **Engineering Coordinator** from various international companies such as the **Fluor Daniel, KBR South Africa, ESKOM, MEGAWATT PARK, CHEMEPIC, PDPS, CAKASA, Worley Parsons, Lurgi South Africa, Sasol, Foster Wheeler, Bosch & Associates, BCG Engineering Contractors, Fina Refinery, Sapref Refinery, Secunda Engine Refinery** just to name a few.

Mr. Frampton has a **Bachelor degree in Industrial Chemistry** from **The City University in London**. Further, he is a **Certified Instructor/Trainer, a Certified Internal Verifier/Trainer/Assessor** by the **Institute of Leadership & Management (ILM)** and has delivered numerous trainings, courses, workshops, conferences and seminars internationally..



Training Methodology

All our Courses are including **Hands-on Practical Sessions** using equipment, State-of-the-Art Simulators, Drawings, Case Studies, Videos and Exercises. The courses include the following training methodologies as a percentage of the total tuition hours:-

- 30% Lectures
- 20% Practical Workshops & Work Presentations
- 30% Hands-on Practical Exercises & Case Studies
- 20% Simulators (Hardware & Software) & Videos

In an unlikely event, the course instructor may modify the above training methodology before or during the course for technical reasons.

Course Program

The following program is planned for this course. However, the course instructor(s) may modify this program before or during the course for technical reasons with no prior notice to participants. Nevertheless, the course objectives will always be met:

Day 1: Sunday, 07th of September 2025

0730 – 0800	<i>Registration & Coffee</i>
0800 – 0815	<i>Welcome & Introduction</i>
0815 – 0830	PRE-TEST
0830 – 0930	Introduction
0930 – 0945	<i>Break</i>
0945 – 1100	Understanding Operational Principles & Why Plants Get Upset
1100 – 1230	Roles & Responsibilities
1230 – 1245	<i>Break</i>
1245 – 1420	Emergency Team Buildings & Responsibilities of Each Member - Case Study
1420 – 1430	Recap
1430	<i>Lunch & End of Day One</i>

Day 2: Monday, 08th of September 2025

0730 – 0900	How Incidents Develop & Common Reasons
0900 – 0915	<i>Break</i>
0915 – 1100	Review of Several Incidents <i>Two Common Causes were Identified that Contributed to those Incidents</i>
1100 – 1230	Improper Management of Manpower During Upset Conditions
1230 – 1245	<i>Break</i>
1245 – 1420	Improper Management of Manpower During Upset Conditions (cont'd)
1420 – 1430	Recap
1430	<i>Lunch & End of Day Two</i>



Day 3: Tuesday, 09th of September 2025

0730 – 0930	<i>Root Cause Analysis (RCA)</i>
0930 – 0945	<i>Break</i>
0945 – 1100	<i>Risk Register</i>
1100 – 1215	<i>Incidents Development Scenarios – Discussion</i>
1215 – 1230	<i>Break</i>
1230 – 1420	<i>Incidents Development Scenarios – Discussion (cont'd)</i>
1420 – 1430	<i>Recap</i>
1430	<i>Lunch & End of Day Three</i>

Day 4: Wednesday, 10th of September 2025

0730 – 0930	<i>Emergency Team Building & Improper Management of Manpower During Upset Conditions</i>
0930 – 0945	<i>Break</i>
0945 – 1100	<i>Improper Risk Assessment of Operation Conditions During Plant Upset</i>
1100 – 1215	<i>Risk Assessment & Risk Evaluation Risk Matrix</i>
1215 – 1230	<i>Break</i>
1230 – 1420	<i>Recognizing Key Points & Controlling Elements in Different Process</i>
1420 – 1430	<i>Recap</i>
1430	<i>Lunch & End of Day Four</i>

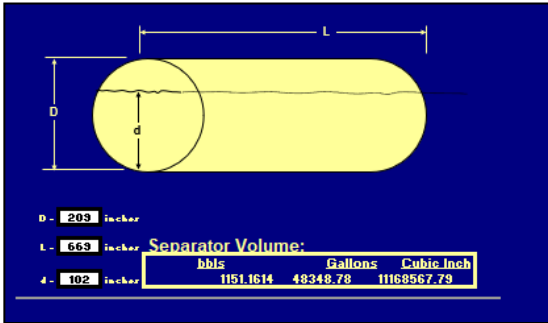
Day 5: Thursday, 11th of September 2025

0730 – 0930	<i>Building Successful Emergency Team & Each One Roles & Responsibilities</i>
0930 – 0945	<i>Break</i>
0945 – 1100	<i>Closing Gaps & Correcting Scenarios</i>
1100 – 1215	<i>Closing Gaps & Correcting Scenarios (cont'd)</i>
1215 – 1230	<i>Break</i>
1230 – 1345	<i>Case Study & Discussion</i>
1345 – 1400	<i>Course Conclusion</i>
1400 – 1415	<i>POST-TEST</i>
1415 – 1430	<i>Presentation of Course Certificates</i>
1430	<i>Lunch & End of Course</i>



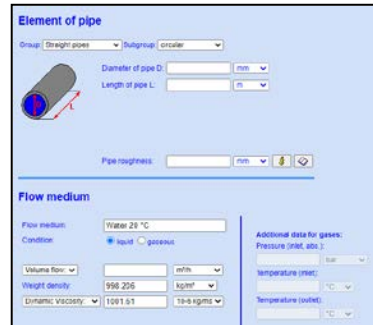
Simulator (Hands-on Practical Sessions)

Practical sessions will be organized during the course for delegates to practice the theory learnt. Delegates will be provided with an opportunity to carryout various exercises using various online system calculator.

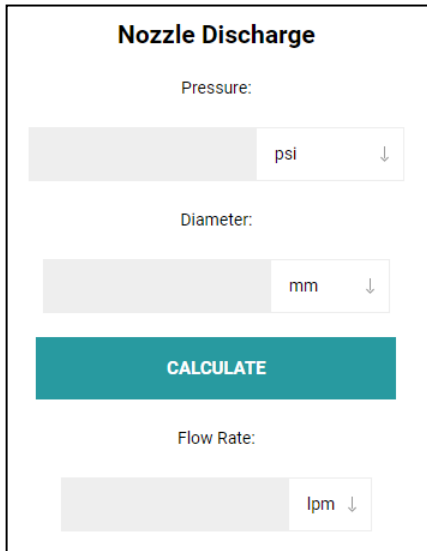


	bbls	Gallons	Cubic Inch
Separator Volume:	1151.1614	48348.78	11168567.79

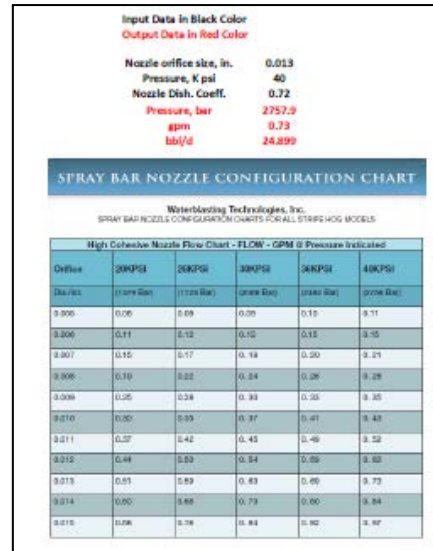
Tank Volume Calculator



Pressure Drop Online-Calculator



Nozzle Discharge



Input Data in Black Color
Output Data in Red Color

Nozzle orifice size, in. 0.013
Pressure, K psi 40
Nozzle Dish. Coeff. 0.72
Pressure, bar 2757.9
gpm 0.73
lbbl/d 24.899

Orifice	20KPSI (1129 Bar)	28KPSI (1129 Bar)	30KPSI (2068 Bar)	38KPSI (2638 Bar)	48KPSI (3379 Bar)
0.006	0.06	0.08	0.09	0.10	0.11
0.008	0.11	0.12	0.12	0.13	0.15
0.007	0.15	0.17	0.18	0.20	0.21
0.008	0.19	0.22	0.24	0.26	0.28
0.009	0.25	0.28	0.30	0.32	0.35
0.010	0.30	0.33	0.37	0.41	0.43
0.011	0.37	0.42	0.45	0.49	0.52
0.012	0.44	0.50	0.54	0.58	0.62
0.013	0.51	0.58	0.63	0.68	0.73
0.014	0.59	0.68	0.73	0.80	0.84
0.015	0.68	0.78	0.84	0.92	0.97

Nozzle Calculator

The horsepower required to adiabatic compression of air can be calculated with the calculator below:

N - number of stages

V - volume flow of compressed air at atmospheric pressure (cfm, ft³/min)

k - adiabatic expansion coefficient

P_2 - absolute final pressure (psi)

Horsepower Calculator



Water Flow Rate through an Orifice Calculator



Convert Cubic Feet Of Natural Gas to Barrels Of Oil Equivalent

Cubic Feet Of Natural Gas

Barrels Of Oil Equivalent (bboe)

Cubic Feet Calculator

Corrosion Rate Calculator

Enter data in given fields and click on Calculate for resultant corrosion rate.

Weight Loss microgm

Density gm/cm3

Area mm2

Time millisec

Result:

Corrosion Rate in mpy

Corrosion Rate Calculator

HYDRONICS CALCULATOR

Water velocity calculator

Water flow rate (gpm) gpm

Pipe diameter (inches) inches

V = ft/min

Minimum pipe diameter calculator

Water flow rate (gpm) gpm

Water velocity (ft/min) ft/min

D = inches

Water flow rate calculator

Pipe diameter (inches) inches

Water velocity (ft/min) ft/min

Q = gpm

Flow Rate (GPM)	Velocity (ft/min)	Pressure Drop (psi/100ft)
10	1.07	0.47
20	2.14	1.88
30	3.21	4.24
40	4.28	7.74
50	5.35	12.37
60	6.42	18.12
70	7.49	24.99
80	8.56	32.98
90	9.63	42.09
100	10.70	52.32
110	11.77	63.67
120	12.84	76.14
130	13.91	89.73
140	14.98	104.44
150	16.05	120.27
160	17.12	137.22
170	18.19	155.29
180	19.26	174.48
190	20.33	194.79
200	21.40	216.22
210	22.47	238.77
220	23.54	262.44
230	24.61	287.23
240	25.68	313.14
250	26.75	340.17
260	27.82	368.32
270	28.89	397.59
280	29.96	427.98
290	31.03	459.49
300	32.10	492.12
310	33.17	525.87
320	34.24	560.74
330	35.31	596.73
340	36.38	633.84
350	37.45	672.07
360	38.52	711.42
370	39.59	751.89
380	40.66	793.48
390	41.73	836.19
400	42.80	880.02

Hydronics Calculator

Pipe Pressure Loss Calculator

Inputs

Pressure at A (absolute) kPa

Average fluid velocity in pipe, V m/s

Pipe diameter, D cm

Pipe relative roughness, e/D mm

Pipe length from A to B, L m

Elevation gain from A to B, ΔZ m

Fluid density, ρ kg/l

Fluid viscosity (dynamic), μ cP

Pipe Pressure Loss Calculator

BTU-Calculator-&-BTU-Formulas-for-Water-Circulating-Heat-Transfer

Weighed Water Test

Measure the flow of water through your process by timing how long it takes to fill a known volume container. For example, allow your process water to fill a 5-gallon container. Accurately measure the water temperature entering and exiting your process. Use this formula to calculate BTU cooling required:

Formula

BTU = Flow Rate In GPM (of water) x (Temperature Leaving Process - Temperature Entering Process) x 500.4*Formula changes with fluids others than straight water.

BTU Calculator for Weighed Water Test

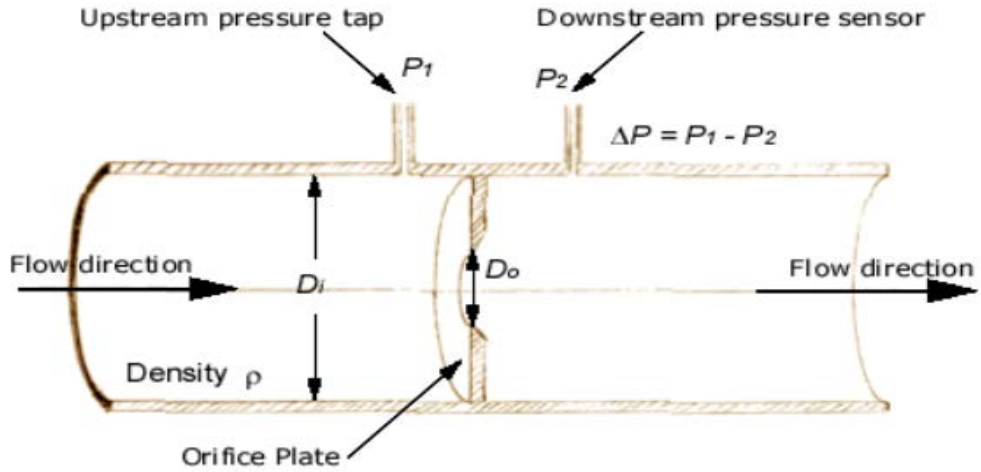
Water Flow Rate In Gallons Per Minute GPM

Inlet Water Temperature To Process °F

Outlet Water Temperature From Process °F

BTU Calculator





Inputs

Pipe (inlet) diameter upstream of orifice, D_i :	8	in ▾
Orifice diameter (less than the inlet diameter), D_o :	3	in ▾
Pressure difference across the orifice, Δp :	20	psi ▾
Fluid density, ρ :	835	kg/m ³ ▾
Flow Coefficient, C_f :	0.82	

Answers

Velocity at the inlet, V_i :	2.10 m/s	m/s ▾
Volumetric Flowrate, Q :	1080 gpm	gpm ▾
Mass Flowrate:	56.7 kg/s	kg/s ▾

Flow Rate through an Orifice or Valve Calculator



Net Positive Suction Head Calculator - In terms of head

Pump Formulas Calculator — Imperial and SI Units

Select a System Units
 Imperial Units SI Units

Ha

Imperial Units Ha = absolute pressure of the suction vessel, ft // SI Units Ha = absolute pressure of the suction vessel, m

Hvpa

Imperial Units Hvpa = fluid vapor pressure at pumping temperature, ft // SI Units Hvpa = fluid vapor pressure at pumping temperature, m

Hst

Imperial Units Hst = static head to suction reference point (usually center line of the impeller), ft // SI Units Hst = static head to suction reference line (usually center point of the impeller), m

Hfs

Imperial Units Hfs = suction line losses, ft // SI Units Hfs = suction line losses, m

NPSH

Imperial Units NPSH = net positive suction head at reference point (usually center line of the impeller), ft // SI Units NPSH = net positive suction head at reference point (usually center line of the impeller), m

Net Positive Suction Head Calculator

Net Positive Suction Head Calculator - In terms of pressure and head

Pump Formulas Calculator — Imperial and SI Units

Select a System Units
 Imperial Units SI Units

Pa

Imperial Units Pa = absolute pressure of the suction vessel, psia // SI Units Pa = absolute pressure of the suction vessel, kPa

Pvpa

Imperial Units Pvpa = fluid vapor pressure at pumping temperature, psia // SI Units Pvpa = fluid vapor pressure at pumping temperature, kPa absolute

Hst

Imperial Units Hst = static head to suction reference point (usually center line of the impeller), ft // SI Units Hst = static head to suction reference line (usually center point of the impeller), m

Hfs

Imperial Units Hfs = suction line losses, ft // SI Units Hfs = suction line losses, m

SG

SG = specific gravity

NPSH

Imperial Units NPSH = net positive suction head at reference point (usually center line of the impeller), ft // SI Units NPSH = net positive suction head at reference point (usually center line of the impeller), m

Net Positive Suction Head Calculator

Input Data in Black Color
 Output Data in Red Color

lbs/gal. 11
 kg/lit. 1.318

Pounds per Gallon	Kilograms per Liter	Conversion Factor
7.0 lb/gal	0.84 kg/l	0.92
8.0 lb/gal	0.96 kg/l	0.98
8.34 lb/gal	1.00 kg/l (water)	1.00
9.0 lb/gal	1.08 kg/l	1.04
10.0 lb/gal	1.20 kg/l	1.10
10.65 lb/gal	1.28 kg/l (28% Nitrogen)	1.13
11.0 lb/gal	1.32 kg/l	1.15
12.0 lb/gal	1.44 kg/l	1.20
14.0 lb/gal	1.68 kg/l	1.30

PPG to KG Calculator

Liquid Pipeline Calculator Software

Inputs

Pressure at A (absolute): 1000 psi

Average fluid velocity in pipe, V: 5.1674 ft/s

Pipe diameter, D: 14 in

Pipe relative roughness, e/D: 0.000357 in/in

Pipe length from A to B, L: 80 km

Elevation gain from A to B, ΔZ: 0 m

Fluid density, ρ: 965.44 kg/m³

Fluid viscosity (dynamic), μ: 5 cP

Liquid Pipeline Calculator

Cv Calculator for Valve Sizing

Calculation type: CV Flow

Medium Type: Liquid Gas

Inlet pressure (P1): PSIA

Outlet pressure (P2): PSIA

Flow rate (Q): SCFM

Temperature: °Fahrenheit

System medium: Acetylene

Specific gravity: 0.907

CALCULATE

Cv Calculator

Find Flow

$$Q = C_d A \sqrt{\frac{2}{\rho} \Delta P}$$

Coefficient: 0.62

Specific Gravity: 0.875

Diameter: mm

Pressure Drop: bar

Flow: lpm

Find Flow Calculator

Inputs

Pipe (inlet) diameter upstream of orifice, D₁: 10 cm

Orifice diameter (less than the inlet diameter), D₂: 8 cm

Pressure difference across the orifice, Δp: 10 Pa

Fluid density, ρ: 1.29 kg/m³

Flow Coefficient, C_d: 0.7

Flowrate Calculator



Coefficient-of-Discharge-Calculator

Calculate discharge coefficient...

using... [hydraulic head](#)

Water level

H

Q

d

Flow parameters

Diameter (d) m

Area (A) m²

Head (H) m

Actual discharge (Q) m³/s

Coefficient Discharge Calculator

Convert horsepower hour to gallon [U.S.] of diesel oil

horsepower hour

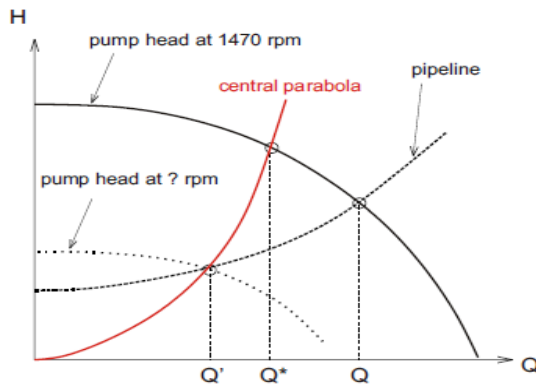
gallon [U.S.] of diesel oil

Horsepower Hour Calculator

Liquid Pumping Program		Output Results	
Input Data		Flow Velocity, ft/s	5.0154
API	28	Erosion Velocity, ft/s	13.440
c.P.	5	E/I.D.	0.001786
1000 bbl/d	3.3	sp.gr.	0.8871
Length, km	2.4384	Re	19290.3
I.D., in.	2.800	F	0.02987
Rough. (E), in.	0.005	Hf, psi	153.67
Difference in elev., m	50	Hf, m water	108.17
Destination press., psi	60	Total Pump Dich. psi	276.68
Pump Suc. psi	80	TDP, psi	196.68
Overall Pump Eff., %	65	Hydr. Power, HP	16.99
Motor Eff., %	90	Hydr. Power, Kw	12.67
Motor Loading %	80	Shaft Power, HP	18.88
		Shaft Power, Kw	14.083
		Nama Plate Motor HP	23.60
		Nama Plate Motor Kw	17.60

A pump running at 1470[rpm] with $H_{pump} = 45 - 2781Q^2$ head delivers water into a pipeline with $H_{pipe} = 20 + 1125Q^2$. Calculate the required revolution number for the reduced flow rate $Q' = 0.05[m^3/s]$.

Solution:



- The actual working point is given by the solution of $H_{pump} = H_{pipe}$, which gives $Q = 0.08[m^3/s]$ and $H = 27.2[m]$.
- Affinity states that while varying the revolutionary speed, H/n^2 and Q/n remain constant. Thus, also H/Q^2 remains constant, let's denote this constant by a . So, while varying the revolutionary speed, the working point moves along the *central parabola* (see figure), given by $H_{ap} = aQ^2$.

However, as Q' is given and we also know that this point has to be located on the pipeline characteristic, we know that $H' = 20 + 1125 \times 0.05^2 = 22.81[m]$. Thus, the parameter of the affine parabola is $a = H'/Q'^2 = 9125$.

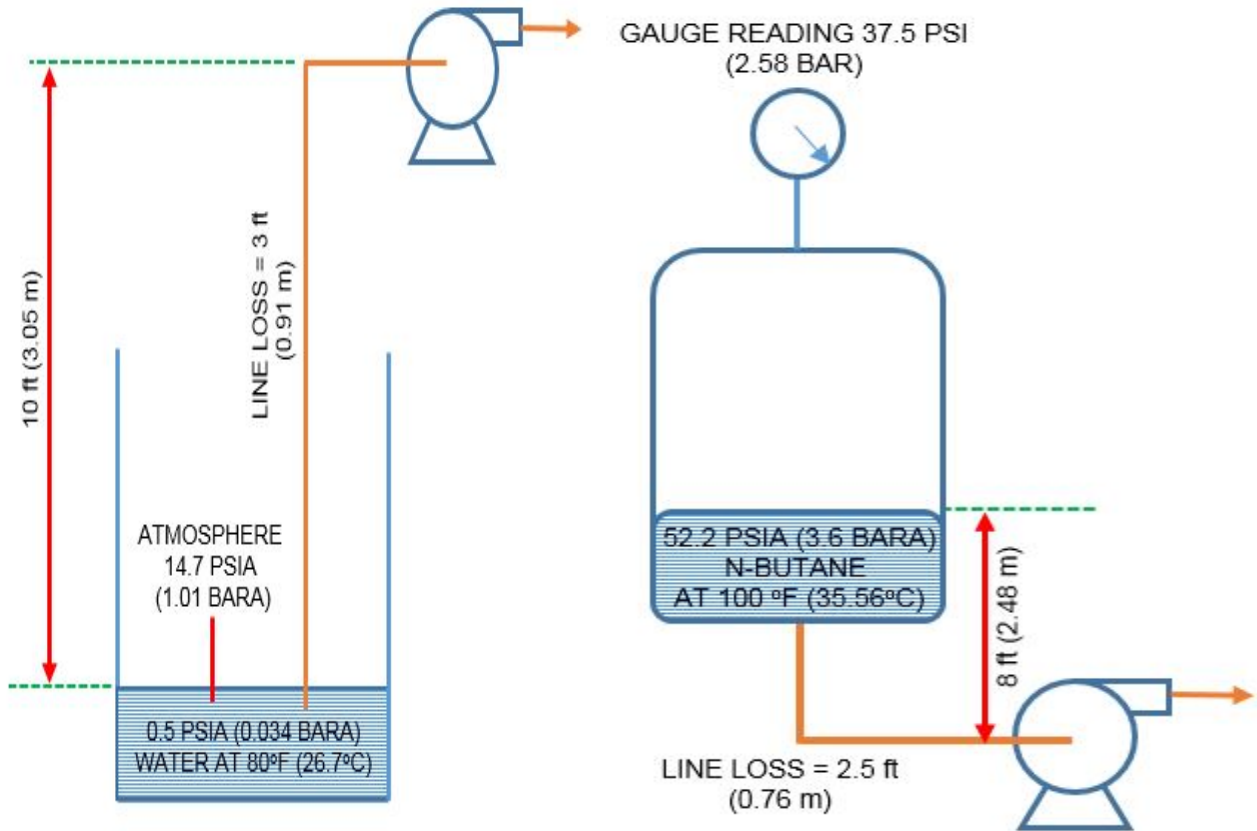
Q^* is given by the intersection of the affine parabola and the original pump characteristic: $H_{ap}(Q^*) = H_{pump}(Q^*)$, which gives $Q^* = 0.06148[m^3/s]$ with $H^* = 34.5[m]$.

Now we can employ affinity between Q^* and Q' :

$$n' = n^* \frac{Q'}{Q^*} = 1470 \times \frac{0.05}{0.06148} = 1195.5[rpm]$$

and just for checking the calculation

$$H' = H^* \left(\frac{n'}{n^*} \right)^2 = 34.5 \times \frac{1195.5^2}{1470^2} = 22.81[m].$$



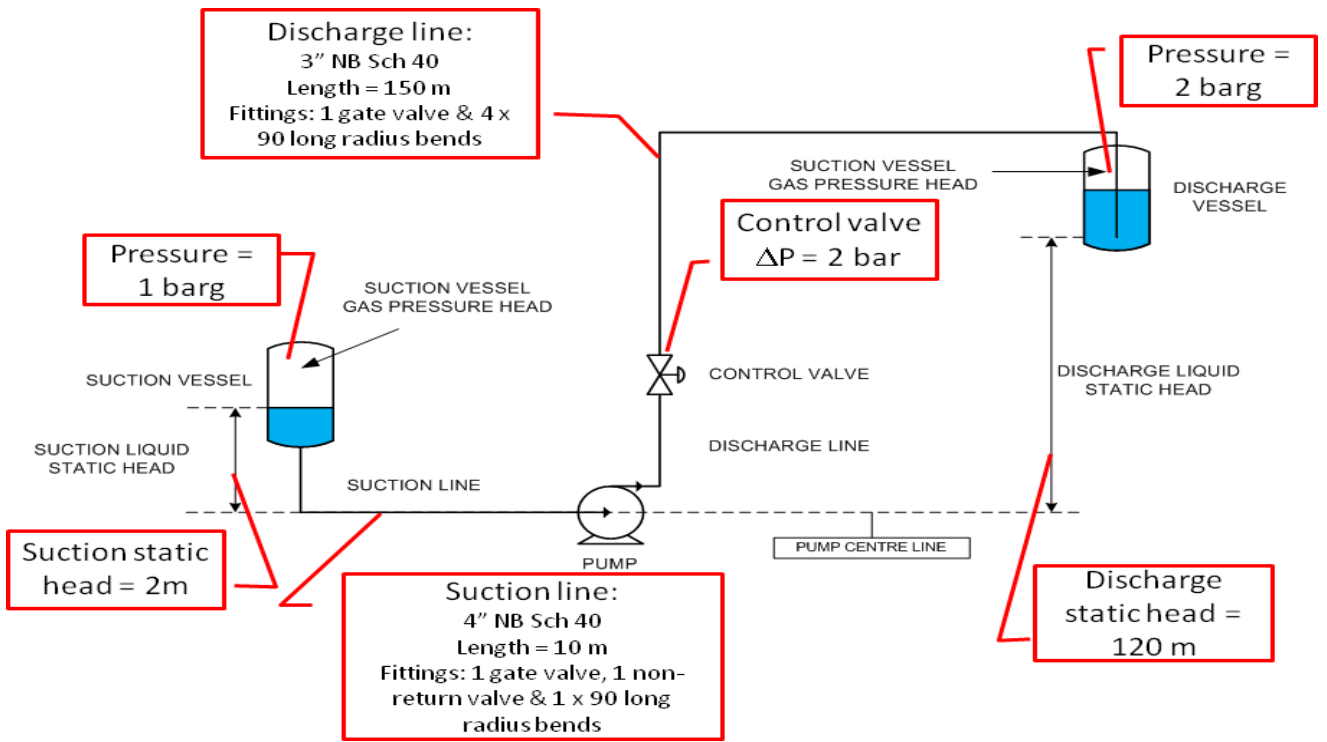
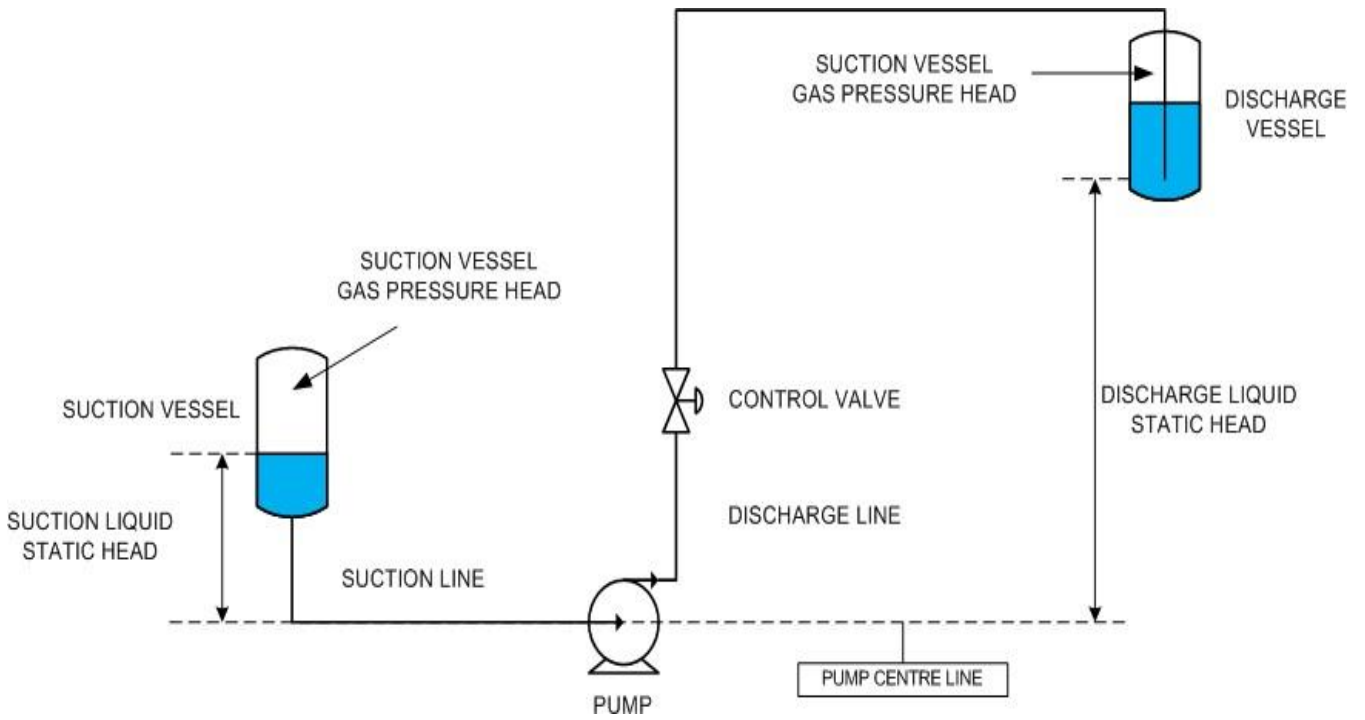
NPSHA of pump – suction lift

NPSHA of pump – at boiling point
SG of n-butane at 100 deg F = 0.56

$$NPSHA = Hatmp. +/- Hs - Hf - Hvap.$$

https://engineeringunits.com/net-positive-suction-head-calculator/?utm_content=cmp-true
<http://www.pressure-drop.com/Online-Calculator/index.html>

NPSH Calculations		Output Results	
Input Data		Flow Velocity, ft/s	
API	36	E/I.D.	0.001671
c.P.	3	sp.gr.	0.8448
Vapor pressure, psi	10	Re	17363.9
Atmp. Pressure, psi	14.7	F	0.0302
Height above pump, ft	20	Hf, psi	0.048
1000 bbl/d	2.0	Hf, ft water	0.111
Length, km	0.003	NPSHA, ft oil	32.72
I.D., in.	2.992	NPSHA, ft water	27.64
Rough. (E), in.	0.005		





Calculator

PUMP DETAILS

Pump tag number		P-001
Suction vessel tag number		V-001
Discharge vessel tag number		V-002
Barometric pressure	P_{atm}	1.013 bara
NPSH available margin	H_{margin}	0 m
Pump efficiency	η	70%

FLUID PROPERTIES

Fluid		Water
Phase		Liquid
Flowrate	m	30000 kg/hr
Density	ρ	998 kg/m ³
Viscosity	μ	1 cP
Vapour pressure	P_{vap}	0.023 bara

VESSEL GAS PRESSURES

Suction vessel gas pressure	P_{suc_vessel}	1 barg
Discharge vessel gas pressure	P_{dis_vessel}	2 barg

STATIC HEADS

Suction static head	$H_{suc_static_head}$	2 m
Discharge static head	$H_{dis_static_head}$	120 m

PIPELINES

		Suction Line	Discharge Line	
Pipe nominal diameter		4	3	inch
Pipe schedule		Sch 40	Sch 40	
Pipe internal diameter	d	102.26	77.92	mm
Pipe length	L	10	150	m
Absolute roughness	e	0.046	0.046	mm

OUTPUTS

Volumetric flow rate Q 30.060 m³/hr

		Suction Line	Discharge Line	
Relative roughness	e:d	0.00045	0.00059	
Flow area	A	0.00821	0.00477	m ²
Velocity	u	1.02	1.75	m/s
Reynolds No.	Re	103758	136170	
Flow regime		turbulent	turbulent	
Friction factor	f	0.02011	0.02010	
Pipe velocity head loss	K_{pipe}	1.966	38.695	
Fittings total velocity head loss	$K_{fittings}$	1.724	2.152	
Frictional pressure loss	$\Delta P_{friction}$	0.02	0.62	bar
Frictional head loss	$H_{friction}$	0.19	6.38	m

Pump suction pressure	$P_{suction}$	2.19 bara
Pump suction head	$H_{suction}$	22.37 m
Pump discharge pressure	$P_{discharge}$	15.39 bara
Pump discharge head	$H_{discharge}$	157.16 m
Net positive suction pressure available	P_{NPSHA}	2.17 bara
Net positive suction head available	NPSHa	22.13 m
Pump total differential pressure	ΔP_{pump}	13.20 bar
Pump total differential head	H_{pump}	134.79 m
Pump absorbed power	E	15.74 kW



Results of above calculations may be confirmed through either of following links:

<https://www.swagelok.com/en/toolbox/co-calculator>

https://experttoolsonline.com/danfoss/orifice_calculator

https://www.efunda.com/formulae/fluids/calc_orifice_flowmeter.cfm

<https://www.omnicalculator.com/physics/coefficient-of-discharge>

Power Calculations: <https://inventory.powerzone.com/resources/centrifugal-pump-power-calculator/%3Aflu%3DGPM%3Apru%3DHEAD%20FT%3Apu%3DHP>

<http://irrigation.wsu.edu/Content/Calculators/General/Required-Water-Pump-HP.php>

Required Compressor Horsepower

https://www.engineeringtoolbox.com/horsepower-compressed-air-d_1363.html

<u>Input Data</u>		<u>Output Results</u>	
T1, F	60	Compression Ratio	34.014
K	1.35	Cp, J/kg/K	1107
P1, psi	14.7	Gas, cfm	36791.50
P2, psi	500	Gas, kg/s	21.250
Gas sp.gr.	1	Theoretical Power, HP	9731.847
No. of Comp. stages	3	Total Required HP	12721.37
Gas million SCMD	1.5		
Eff. of Gas Comp., %	85		
Eff. of Driving Motor, %	90		

Heater Duty

<https://www.advantageengineering.com/fyi/288/advantageFYI288.php>

<u>Input Data</u>		<u>Output Results</u>	
		Delta Temp., C	15.6
		Mega Watt	0.220
		Billion Joule/hr.	0.791
Million BTU/hr.	0.75	gpm	25.0
API	10.0	gallon/hr.	1498.4
Specific Heat, BTU/lb/F	1.00	Lit./min.	94.5
Delta Temp., F	60	m3/hr.	5.7
Heater Eff., %	100	1000 bbl/d	0.856
		Required Diesel Lit./day	502.90
		Required Diesel bbl/d	3.16
		Required Gas, 1000 ft3/d	16.364
		Required crude oil, bbl/d	3.268

<https://www.enggcyclopedia.com/2011/09/problem-solving-heat-exchanger-tubeside-pressure-drop-calculation/>





<u>Input Data</u>	<u>Output Results</u>		
Mass Flow Rate, kg/hr.	2000.0	cm ³ /s	562.303
Fluid Density, Kg/m ³	988.0	V, cm/s	110.9720
Visc., c.P.	0.53	Re	52544.59
Pipe Diameter (D), in.	1	f	0.0261
Roughness (E), mm	0.045	Total Hf, cm (per single tube)	22.5583
Tube Length, m	3.5	Total Hf, psi (per single tube)	0.3166
No. of tubes	1	Total Hf, bar (per single tube)	0.0218

Heat exchanger tube side pressure drop calculation

Calculate the tube side pressure drop for the following heat exchanger specification,

Process fluid = water

Inlet pressure = 4 barg

Inlet temperature = 50^oC

Outlet temperature = 30^oC

Tubeside flowrate = 50000 kg/hr

Number of tubes = 25

Tube ID (internal diameter) = 1 inch

Tube length = 3.5 m

Total volumetric flow = 50000 kg/hr ÷ 988.0 kg/m³ = 50.61 m³/hr Volumetric flow in each 1" tube = 50.61 ÷ 25 = 2.02 m³/hr Pressure loss per unit length of the tube is then calculated using [EnggCyclopedia's pressure drop calculators for pipes and tubes](#). This calculator is based on [Darcy-Weisbach equation](#).

Pressure loss across a single tube ($\Delta P/L$) = 6.17 bar/km

SINGLE PHASEFLOW INPUTS

W – Mass flow capacity kg/h

ρ – Density of fluid kg/m³

μ – Viscosity of fluid (either liquid or gas) cP

PIPE SPECIFICATIONS

e – Effective roughness of the pipe mm

d – Nominal diameter of the pipe inches

sch – pipe schedule

RESULTS

Fluid Velocity m/s

Volumetric flow m³/hr

Reynold's No.

Pressure loss bar/km

Tube length (L) = 3.5 m

Tubeside pressure drop (ΔP) = 6.17 × 3.5 / 1000 = 0.0216 bar





Another alternative is to directly use EnggCyclopedia's Heat Exchanger Tube side Pressure Drop Calculator. All the inputs given in the sample problem statements are given to the calculator and pressure drop across the tubeside is calculated as output. This calculator uses the same basic steps discussed above and hence the answer also matches with the figure above (0.0216 bar) . The following image is a snapshot of this direct calculation of tubeside pressure drop.

Exchanger tubeside pressure drop

Tubeside inputs

Total tubeside <u>mass</u> flow	<input type="text" value="50000"/>	kg/hr
Tubeside <u>Density</u>	<input type="text" value="988"/>	kg/m ³
Tubeside <u>Viscosity</u>	<input type="text" value="0.53"/>	cP
Number of tubes	<input type="text" value="25"/>	
Total tube length (accounting for all tube passes)	<input type="text" value="3.5"/>	m
Tube nominal diameter	<input type="text" value="1"/>	inches
Tubeside roughness	<input type="text" value="0.045"/>	mm
<input type="button" value="Calculate pressure drop"/>	<input type="button" value="Reset"/>	

Results

Tubeside pressure drop bar

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